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Hydraulics of Closed Conduit Spillways

Part XII:

The Two-Way Drop Inlet with a Flat Bottom

ARS-NC-14 September 1974







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Preface

This publication, the 12th part of a group of publications dealing with the hydraulics of closed conduit "pilmyay, reports tests on the two-way drop inlet. The previous 11 parts were published as technical paper unifor the major title "Pilyraulics of Closed Conduit Spillways" by the St. Anthony Pilal Fordraulic Laboratory (SAPHL), University of Minnesota, Minnesota, Minnesota, Minnesota, Startenburghia, The earlier publications are:

Part I. Theory and Its Application, by F. W. Blaizdell. SAFHL Tech. Paper No. 12, Ser. B, 22 pp., illue, Jan. 1952 (rev. Feb. 1958). Gives theory, symbols, and bibliography.

Parts II through VII. Results of Tests on Several Forms of the Spillway, by F. W. Blaisdell. SAFHI. Tech. Paper No. 18, Ser. B., 50 pp., illus, March 1988. Parts II through VI describe the hydraudic performance and present discharge coefficients for five forms of the closed conduit spillway; Part VII discusses vortices and their offect on the sailway canacity.

Part VIII. Miscellaneous Laboratory Tests; Part IX. Pield Tests, by F. W. Blaisdell. SAFHL Tech. Paper No. 19, Ser. B, 54 pp., illus., March 1958. Reports tests on models of specific field structures and on field structures themselves.

Part X. The Hood Inlet, by F. W. Blaisdell and C. A. Donnelly. SAFHL Tech. Paper No. 20, Ser. B, 41 pp., illus., April 1958. Reports the development of the hood inlet.

Part XI. Tests Using Air, by F. W. Blaisdell and G. G. Hebaus. SAFHL Tech. Paper No. 44, Ser. B, 53 pp., illus., Jan. 1968. Discusses the use of air for tests of closed conduit spillways.

Summary

This report presents the results of experiments on a rectangular drop finish baving a width equal to the barrel diameter, a fish thorizm, and afts, borizontal antivortex plate supported above the drop intel creek by extensions of the drop intel endwalls. This structura is called a two-way drop intel because the water enten over only the two sides of the rectangular drop intel. The experiments were conducted as a generalized study, that is, all dismensions are expressed in terms of the pipe dismeter and the results are applicable to any size structure table is generalized within y the structure tested.

Two separate apparatus were used to expedite the test program. In one, the test fluid was water; in the other, it was air. Inlet capacity, pipe priming, and vortex phenomena were studied using water. Entrance less and pressure coefficients for various linlet proportions were studied using air, since full pipe flow tests can be conducted more advantageously with air than with water.

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Part XII:

The Two-Way Drop Inlet with a Flat Bottom

BY CHARLES A. DONNELLY, GEORGE G. HERAUS, AND FIRST W. BLAISDELL²

Introduction

The two-way drop inlet is used as an entrance to closed conduct spillways. The drop inlet, shown in figure XII-1, is rectangular. Weter enters the inlet over the two ides that are parallel to the pipe axis, hence the name "two-way drop inlet." The endwalls are extended upward and outward to support a flat, horizontal antivortes plate over the drop inlet cross. Trash racies can be conveniently mounted on the extended endwalls and outer edges of the nativortes plate and the drop inlet has not been appeared to support a flat, horizontal antivortes plate over the drop inlet has the drop inlet has the drop inlet has not been appeared to support a flat, horizontal antivortes plate can be conveniently mounted on the extended endwalls and outer edges of the antivortex plates of the nativerse has the support of the drop in the convenient of the convenient

The two-way drop inlet is used by the Soil conservation Service (SGS) as a closed conduit spillway extrance for upstream flood prevention, which was the service of the ser

drop inlet. On the other hand, if economics dictates that a two-stage inlet be used, low-stage orfices can be incorporated into the drop inlet walls. The two-way covered drop inlet concept originated with M. Culp, now clift, Design Branch, Engineering Division, SCS, when he was head of the Engineering Studied Unit in 1951. The study reported here refines and expands the design concept.

³ Research by the St. Anthony Falls Hydraulie Laboratory Conservation Structures Investigations Unit, Agr. Res. Serv., U.S. Dept. of Agr., in cooperation with the Minn. Agr. Exp. Sta. and the St. Anthony Falls Hydraulic Lab. Univ. of Minn., Minnespolis.

Drop Inlet Description

The two-way drop inlet is illustrated in figure XII-1. The drop inlet models were made of transparent plastic. For the water tests, the burrel inside diameter D was 2.25 inches. D was 3 inches for the air tests.

The drop inlet width W was equal to the barrel diameter for all tests.

The drop inlet height Z, was 5D for all air tests. Heights of 5D, 4D, 3D, and 2D were used in the water tests to determine the minimum height at which the pipe would prime satisfactorily.

Drop inlet lengths 8 of 1D, 1.5D, 2D, 3D, 8D, and 10D were tested using both water and air. Most of the drop inlets were built 10D long. Shorter inlets were formed by inserting a faise wall into the drop inlet at the proper location to give the desired length.

The bottom of the inlet was flat and horizontal. Crest thicknesses t, of 0.111D and 0.444D were used in the water tests, and thicknesses of 0.104D.

used in the water tests, and thicknesses of 0.104D, 0.157D, 0.242D, 0.331D, 0.500D, and 0.658D in the air tests. In all tests the outside edge of the creat was square. The inside of the crest had a radius r of one-half the creat thickness as shown in fewer XIL-1

The antivortex plate overhang \mathbb{L}_{\bullet} is measured outward from the outside of the drop inlet as shown in figure XI-I. The length of the entitortex plate is the same as the drop inlet length B. The antivortex characteristics of the plate were determined in the water tests for plate overhangs of 0.4D, 0.6D, 0.8D, 1.0D, 1.8D, 2D, 3D, and 4D. The effect of plate overhang on the loss coefficients.

cient was evaluated in the air tests for L, values of 0.295D, 0.506D, 0.753D, 1.00D, 1.41D, 1.5D, 2D, 3D, and 4D.

The antivortex plate height Z₂ is the distance from the drop inlet crest to the bottom of the antivortex plate. A recommended range of values of Z₂ over which good inlet performance pressure is the property of the contract of the contract of the heights aranging from 0.10 to 1.40. Neminal plate heights of 0.10, 0.20, 0.40, 0.60, and 0.80 were used in the air experiments.

The sine definition—the vertical drop divided by the slope length—of barnel slope S was used. All water tests were conducted at a slope of 20 percent. Slopes of 0, 2.5, 5, 10, 20, 30, and 40 percent were used in the sir tests.

For the water tests, the barrel entrance was souare edged. For the sir tests, both a squareedged and a grooved entrance were used. Two types of square-edged entrances were used. The square-edged barrel inlet shown in figure XII-1(a) and labeled "a" in table XII-1 had its invert at the level of the drop inlet bottom and its crown in the plane of the downstream endwall. The square-edged barrel inlet shown in figure XII-1(b) and labeled "b" in table XII-1 had its invert at the level of the drop inlet bottom but its crown was offset D/8 in the plane of the inlet face from the downstream endwall. The details of the grooved entrance labeled "c" in table XII-1 are shown in figure XII-1(c). The dimensions closely correspond to the American Society for Testing Materials Specifications C 75-41 and C 76-41 for reinforced concrete pipe.

Test Apparatus, Test Procedure, and Analytical Methods

The apparatus, test procedure, and analytical methods used with the water experiments were essentially the same as those described in Part X^a

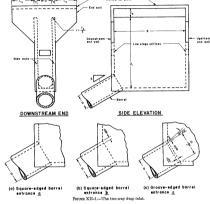
of this series of technical papers. The air apparatus and testing procedures have been described in Part XI^a.

Test Data

The test data are summarized in table XII-1. Included are the series number, the prefixes W. denoting tests with water, and A. denoting tests with air; the geometry of the two-way drop inlets tested; pertinent notes on the performance of the drop inlet; the observed severage energy loss co-

*See references listed in Preface for complete citation.

efficients for the drop inlet creat K, the harrel entrance K, and the complete entrance K, for each series for which they were determined, energy loss coefficients computed using the equations and rules developed from the tests: the percentage deviation of the computed coefficients from the experimentally determined average coficients; and the ratios of the local pressure head



the spillway performance and indicated that some deviations from the hydraulic grade line h, to the feature of the drop inlet was not satisfactory. velocity head in the conduit has at the crown and

invert of the conduit D/2 from the drop inlet. Some of the Notes in table XII-1 require

explanation. Many of the water tests were made primarily to determine the spillway performance. The presence or absence of vortices was used as a measure of the performance of the antivortex wall. The Notes, "p. No vortices," "s, Small vortices," "t, Weak vortices," or "r, Surface vortices" indicate that the vortices had no measurable effect on the spillway performance. The presence of "q, Vortices" or "n, Strong vortices" adversely affected The Note, "v. Orifice flow at crest," indicates

that the opening between the antivortex plate and the crest acted as an orifice to control the flow, and that the height of the antivortex plate above the crest was so low that the spillway performance was not satisfactory.

A surface tension depresoant was added to the water for series W-311 through W-314 to determine the effect of surface tension on the spillway performance. The Note in table XII-1 is "o. Surface tension test."

The Note, "v, Inlet vibrates," indicates that Test continues on page 11.

Tanis XII-1,—Summary of tests Water tests (W): D=0.1875~ft. Where tests (W): D=0.1875~ft. We D; f/D=110.21 Air tests (A): D=0.249~ft. W = D; f/D=137.

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some feature of the inlet caused the flowing water to produce severe vibrations. This was considered unsatisfactory performance

Series W-496 through W-510 were made with reduced drop inlet heights to determine the minimum permissible height. The Note, "z, Orifice flow at burrel entrance," indicates that the harrel entrance acted as an orifice, that the barrel did not fill, and that the performance was unsatisfactory. The Note, "ac, Barrel primed," means that the barrel filled and that the inlet performance was satisfactory.

The Note, "m, Poor hydraulic gradeline in the barrel," means that the piezometric pressures poorly defined the slope of the pressure gradeline in the barrel.

The Note, "n. Condensation during some runs may have caused a change of state" indicates

Test Results

The proportions of the two-way drop inlet that give satisfactory hydraulic performance were first determined. Subsequent tests were made to determine the weir flow capacity, the slug and mixture flow capacity, and the full-flow energy loss that water may have condensed in the barrel This indicates a change of state of the air which would invalidate the basic airflew equations used

in the analysis. The results of these runs are therefore, of questionable value.

The Note. "x, Noisy, vibrating inlet," indicates that poor flow conditions were affecting the performance of the spillway. The observed pressures fluctuated arratically when the inlet was noisy Geometrically similar water models showed vortices, which suggests that vortices also were present in the sir models

The conclusions presented in this publication are based on approximately 7,900 individual tests This figure is estimated from the 20 to 24 test runs made for each of the 256 water series and from 3 to 7 runs for each of the 456 air series listed in table XII-1

and pressure coefficients used to compute the

capacity of and pressures in the spillway. The

performance, the capacity, and the pressures will

be discussed in separate sections.

Spillway Performance

Satisfactory performance of closed conduit spillways requires that (1) the head-discharge relationship be unique and reversible, (2) the structure prime smoothly and positively, and (3) harmful vortices he prevented. In each test the moment of incipient slug flow in the barrel was recorded. Any evidence of vibration in the structure was also noted. Visual observation, judgment, and careful analysis of the data were used to determine if the performance of the drop inlet was satisfactory.

Minimum drop inlet beight and length

The minimum height of the two-way drop inlet that would insure priming of the barrel was determined. Drop inlet heights tested were 2D, 3D, 4D, and 5D. Lengths were 1D, 1.5D, 2D, 3D, and 5D. The width was 1D and the barrel slope was 20 percent for all tests. Endwalls were installed to a height of 0.8D above the crest of the drop inlet. No antivortex plate, however, was installed, except for the 10-long drop inlet, because the barrel primed before the headpool water surface elevation reached the antivortex plate elevation. The antivortex plate, therefore, could have no effect on the priming. Although the head and discharge were measured, notes taken during the tests were used in evaluating the performance of the spillway.

The performance of the 1D-long drop inlet was unsatisfactory at all heights. Air sucked into the drop inlet during slug flow caused severe vibrations of the drop inlet and antivortex plate. Strong vortices were also observed after submergence of the antivortex plate,

The performance of the 22-high drop inlet was unsatisfactory at all lengths. For this height of drop inlet, orifice flow at the barrel entrance alternated with slug flow in the barrel. This caused the headpool to build up during orifice flow and draw down during pipe or slug flow or both. Therefore, the performance of the 22-high drop inlet does not meet the criteria proviously satablished.

Satisfactory performance was obtained for twoway drop inlets 3D, 4D, and 5D high having lengths of 1.6D or greater. Unique head-discharge curves were obtained and vortices were fully

The results of the tests to determine the minimum height of the two-way drop inlet show that the height must be 3D or more and that the length must be 1.5D or more to insure satisfactory performance.

Minimum antivortex plate height

The authors reasoned that the authorise plate must be high enough above the drop infect creat so that the weir flow discharge over the creat would be sufficiently greater now. They place the control of the sufficient plate that the plate that the plate that the barrel would eventually price as the discharge increased. Also, if the authorites plate was too low, the flow into the drop infet was governed by the ordice formed by the authorites of the plate was the plate that the plate

The two-way drop inlet head-discharge test data and notes were examined to determine the maximum discharge at which aluge did not form and the minimum discharge at which alugs were observed. The harrel slope was 20 percent for these tests.

For two-way drop inlets 2D or more in length, the onset of a slug flow was observed to be $Q/D^{3/2} \lesssim 5.1$. For the 1.50-long drop inlet, the onset of slug flow was at $Q/D^{3/2} \lesssim 7.0$. Therefore, the minimum permissible height of the authority of the slug drop way at $Q/D^{3/2} \lesssim 7.0$. Therefore, the minimum permissible height of the authority alpha above the two-way drop indischarges.

The minimum permissible antivortex plate height based on the criteria presented in the preceding paragraph $(Q/D^{8/2} = 7.4 \text{ for } B/D = 1.5 \text{ and } Q/D^{9/2} = 5.1 \text{ for } B/D \ge 2)$ has been computed for five lengths of the drop inlet and for

TABLE XII-2 ... Minimum antivortex plate height

B D	(Z _r /D) minimum	
	C = 3.1	C = 4.0
1.5	0.85	0.72
2	.65	A7
3	.42	.36
6	.30	.25
10	.19	.16

two values of the weir discharge coefficient C in equation I-2*. The results are presented in table XII-2.

Vortex Control

Vortex control means the suppression of those vortices that are of sufficient also and intensity to reduce the spillway capacity. Surface vortices may still form, but it is not necessary to attempt to control them if they have no adverse effect on the spillway capacity. A horizontal plate supported above the crest of the two-way drop inlate is a good antivortex device. However, to be fully effective the antivortex device flower, to be correctly and be of proper size.

Maximum antivortex plate helght

Vortex formation is strongest just after the bogiming of full pipe flow. This occurs at heads and discharges just exceeding those at the intersection of the extended well flow curve and the full pipe flow curve on the head-discharge diagram shown in figure XII-2. The headpool surface elevation at the intersection of these two curves has been selected as the maximum elevacurves has been selected as the maximum elevation of the selection of the selection of the selection of the selection of the selection of the would permit vertices to form under the plate and thus reduct it ineffective.

Minimum antivortex plate overhaug

The length of overhang of the antivorter plate in figure XIII—is an important dimension with regard to the ability of the antivorter plate to control vortices. If the overhang is too short, vortices that adversely affect the spillway capacity will form. If the overhang is too great, the economy of the structure will be adversely affected. Therefore, tests were made to determine

^{*}See reference to Part I in Preface for complete citation.

the minimum overhang of the antivortex plate that would adequately control vortex formation.

The minimum antivortex plate overhang was determined by installing an excessively long own-hang and progressively shortening it until vortices developed that affected the splitway capacity. The Stevens type M water level recorder described in Part X was used to determine the effect of vortices. Charta similar to that shown in Figure X.4 www to obtained for each length of over-heary. If there was vortex action, the full flow in the control of the property of

The discharge through a two-way dron inlet

spillway is, or can be, successively controlled by the two weirs on opposite sides of the drop indet, by the antivortex plate, and by the spillway flowing completely full. The quantitative effect of each of these controls on the spillway capacity will be considered in turn.

Typical head-discharge curves for several antivortex plate heights are shown in figure XII-2.

Weir Flow The flow is controlled by the drop inlet crest acting as a weir when the headpool surface eleva-

tion lies between the elevations of the drop inlet crest and the bottom of the antivortex plate or the intersection of the weir and pipe flow headdischarge curves, whichever is lower.

The tests to determine the relationship be-

See reference to Part X in Preface for complete citation.

Table XII-3.—Minimum antivortex plate overhans

8/D	(L/D) minimum	
1.5	1.0	
2	.6	
3	.5	
5	4	
10	.1	

antivortex plate overhang were tested if the tests would not contribute useful information.

The results of the tests are summarized in table XII-S. It can seen that the longest overhang is required for the shortest drop inlet. The reason for this is that the velocities approaching the creet are greatest and the vortex is strongest, therefore the overhang needed to control the vortex is longest. The height of the antivotte plate above the drop inlet creet had no effect on the minimum overhang required.

The minimum lengths of overhang listed in table XII-3 or considerably less than the length of overhang required to support trash rack: Rigoverned by the permissible velocity through it, and this area ordinarily cannot be obtained unless the overhang length is in excess of the values given in table XII-3. Using lengths of overhang exceeding the minimum is permissible.

Spillway Capacity

tween the weir flow head and the discharge produced erratic results. All attempts to develop reliable equations to predict this relationship were nonproductive. As a result, it is felt that the designer can achieve as a satisfactory predictions of the weir flow discharge using published equations as he could obtain using equations that might be developed from the data obtained during the present experiments.

Antivortex Plate Flow

The horizontal antivorter plate supported above the drop inlet creet causes tha two-way drop inlet closed conduit spillway to set like a self-regulating slopen. Consequently, as the flow increases, the headpool water surface arrains at approximately the same elevation from the time the water surface first bondes the underside of the antivortex plate until the spillway flows completely inil.

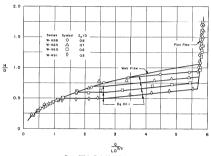


FIGURE XII-2.-Typical head-discharge curves.

The self-regulating siphonic action is a result of slugs in the barrel which cause a partial vacuum in the drop inlet during plate flow. The head causing flow through the two rectangular orifices created by the crest, the antivortex plate, and the endwalls is increased by this partial vacuum. The increased flow required by this increased head, if not supplied by an increased waterflow into the headpool, is supplied by air being sucked into the drop inlet under the antivortex plate. The amount of air entering the spillway depends on the number and frequency of the slugs in the barrel, or the vacuum created by the harrel flowing full of a mixture of water and air. As the waterflow increases after the water first touches the antivortex plate, the airflow will increase to a maximum. and then it will decrease to zero when the spillwav becomes full of water alone. The headpool level increases slowly or not at all as the waterflow increases during the period of airflow.

The problem was to determine the headdischarge relationship and the factors that affect this relationship string the period of airflow. The control measured that the criteria for minimum control measured that the criteria for minimum and maximum Peter performance 'have been met. The data used included 227 different combinations of drop inlet length, antivortey plate beight, antivorter plate overhang, and creat thickness. All drop inlets were 60 high and from 10 to 100 long. The plate beights ranged from 0.105 to 100, 100, the plate towning ranged from 0.135 to 40. Co. the plate towning ranged from 0.135 to 40. The equation developed as a result of the analysis of the data's

$$\frac{H}{g} = \frac{Z}{h} - \left\langle \frac{\partial f}{\partial p} - \partial f \log_{\theta} \frac{L}{h} \right\rangle + \left\langle \partial f - \partial \sigma \frac{L}{p} \right\rangle \frac{Q}{H^{2} G^{2}} \approx \frac{Z}{h}^{2} \quad (\text{XII-1})$$
where the quantities in pointed brackets are zero for negative values. Expressed another way, . . .

and
$$\left\langle \frac{0.1}{3/D} - 0.1 \log_3 \frac{L_1}{D} \right\rangle \ge 0$$

 $0 \le \left\langle 0.1 - 0.05 \frac{L_1}{D} \right\rangle \le 0.1$

that is, $0 \le L_0/D \le 2$. It can be seen that for values of L_0/D greater than 2 the last term in equation XIII-d diseppears and the head does not increase as the discharge increases. As a result, the headpool level does not change during slug and mixture flow when $L_0/D \ge 2$.

A restriction on equation XII-1 is that H/D be not less than Z_p/D. This is because plate flow cannot control the discharge unless the headpool water surface touches the underside of the antivortex plate.

Equation XII-1 is shown in figure XII-2 for several heights of the antivortex plate. The excellent agreement of equation XII-1 with the test data plotted in figure XII-2 is apparent. The agreement shown is better than was obtained in many cases, but the agreement was considered satisfactory for all the data chairmed.

Surface tension effect

The effect of aurine tension on the plate flow and discharge relationably (equation XII-1) was determined. This was done become the nurse of the control of

Four sets of duplicate tosts were made to determine the effect of surface tension. One set of tests used water having a measured surface tension of 71 dynes per continuents. Arravious added to the water for the other set of tests reduced the surface tension to 28 dynes per continuents. This reduction of the surface tension to 57 year. This reduction of the surface tension to 57 year. Grant of the surface tension continuents of the surface tension that the continuents of the surface performance and capacity of the towards drop intel.

Typical results of the tests are shown in figure XII-3 where it can be seen that identical results, within the limits of precision of the experiments, were obtained even for the large change in surface tension. Equally good comparisons were obtained for the other tests.

Pine Flow

Pipe flow denotes the condition that the spillway is flowing completely full of water. There is no flow of air. The head-discharge relationship for pipe flow is given by equation I-59.

Tests were made to determine the entrance loss coefficient K, in equation I-5. Both to expedite the tests and to facilitate the use of the test results by the designer, the entrance loss coefficient was separated into a crest loss coefficient K. and a barrel entrance loss coefficient K. The crest loss coefficient includes the energy losses between the headpool and the midbeight of the drop inlet. The barrel entrance loss coefficient includes the energy losses between the midheight of the drop inlet and the barrel entrance plus those energy losses in the barrel caused by the harrel entrance. This facilitates the design of a two-way drop inlet by making it possible to choose the creet geometry and the barrel entrance geometry independently. The method of analysis permits the loss coefficient for each to be determined separately and then combined to obtain the total entrance loss coefficient for the complete drop inlet.

The effects of the several parts of the two-way drop inlet on each of the energy loss coefficients will be discussed separately.

Crest loss coefficient?

The crest energy loss coefficient K, describe the energy loss that course between the headpool surface and the midheight of the drop inlet. A single piezonetic was used to measure this loss, and per piezonetic was used to measure this loss, and the piezonetic was located on the draw piezonetic was located on the draw piezonetic was similarly located for on the draw piezonetic was similarly located for the water tests only for the 20-long drop inlet. Thus, the air tests were be primary source of information for determining the crest loss conformation for determining the crest loss conformation for determining the crest loss contents to the conformation of coefficient in defined to the conformation of coefficient is defined by the coefficient is defined by the coefficient is defined by the coefficient is defined by the coefficient is defined to the coefficient is de

$$K_s = \frac{h_s}{V^2/\theta_{co}}$$
 (XH-2)

⁴See reference to Part I in Preface for complete citation.

For a theoretical derivation of the creet less and a comparison with experimental data see Hebaus, George G. Creat lesses for two-way drop inlet. Amer. Soc. Givil Ragin. Proc., J. Hydraul. Div., vol. 95, (HY3): 919-940, May 1989.

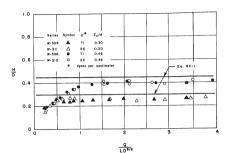


FIGURE XII-3.-The effect of surface tension on the head-discharge curve.

where h_i is the loss in head between the headpool surface and the midheight of the drop inlied feet, $V_i = Q/h_i$ is the average velocity in the drop inlied or riser in feet per second, Q is the rate of flow in cubic feet per second, A_i is the area of the drop lied to riser in aquare feet, and g is the area of the acceleration of gravity in feet per second per

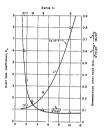
Because the creat loss coefficient K, varied with the drop intel length, the antivortex plate beight, and the creat thickness, the effect of each will be the creat thickness, the effect of each will be vortex plate owntains, will be shown. As expected, the type of barrel entrance and the slope of the barrel had no effect on the creat loss coefficients. Coulty as measured by the Reynold's number had an effect on the creat loss coefficient. The section will conclude with a description of the method and the country of the creat loss coefficient.

Effect of drop inlet length.—The effect of the relative length of the drop inlet B/D on the crest

loss coefficient K_e is shown by the typical curve presented in figure XII-4. The coefficient increases rapidly as the relative drop inlet length in

creet head loss.
$$\frac{k_c}{248-24} = \frac{K_c}{18724}$$
(XII-3)

The dimensionless creat head loss data for the creat lose coefficient presented in figure XII—4 are also plotted there. The data and curves that represent the data show that the actual creat head loss for any yiern discharge and harred diameter decreases with an increase in the relative length of the drop inlet even though the creat lose coefficients increase with relative drop inlet length. The reason the actual creat loss is low



Relative drap tolet length, S./D.

FIGURE XII-4.—The effect of relative drop inlet length on the crest loss coefficient and the dimensionless crest head loss, $b/D \simeq 0.104$, $Z_0/D \approx 0.6$.

even though the coefficient is high for the long drop inlets is that the long drop inlets have a comparatively large area, low velocity, and low velocity head. When the low velocity head is multiplied by even a large crest loss coefficient, the head actually lost at the crest is low.

The dimensionless crost head loss decreases rapidly as the relative drop inlet length increases for relative drop inlet length increases for relative drop inlet lengths Plo loss than about 5. The curve decreases slowly with further increases in the relative drop inlet length. This suggests that there is an optimum inlet length that the designer may wish to determine for each two-way drop inlet.

Effect of antivortex plate height.—The relative height of the flat antivortex plate above the drop inlet creat Z_s/D has an important effect on the creat loss coefficient K_s. This effect is shown in figure XII-5. (The effect of plate overhang, also shown in figure XII-5, will be discussed in the following section.)

The crest loss coefficient is large when the plate beight is low because the high velocity between the plate and the drop inlet crest causes high energy losses. The crest loss coefficients decrease with increasing plate height and approach a constant value.

The autivortex plats height selected by the designer will probably be determined by the performance criteria given previously or by other considerations rather than by the effect of plate height on the creat loss coefficients. However, figure XII-5 shows that the higher the autivorter plate is above the drop pinks creat, the lower will be the head loss at the drop inject creat.

Effect of antivortex plats overhaug.—The railbutte overhang of the antivortex plats boyen the studied of the drop inlet creet 1.07 was found to entailed of the drop inlet creet 1.07 was found to the control of the control of the control of the first is about the control of the control of the curve has been drawn to represent the data oflated for relative overhangs ranging from 0.3 to 4.0. While there is some spread in the data, control and of this present in product of the control of the cont

The open squares, erect triangles, circles, and circled crosses and their solid counterparts plotted in figure XII-5 require explanation. The data represented by the open symbols were obtained using endwalls extending to the edge of the antivortex plate overhang. It can be seen that the open symbols deviate from the curve at the highest plate heights, especially for the shorter drop inlet lengths. The data represented by the solid symbols were obtained using endwalls extending outward 1.5D from the drop inlet wall. They show that lengthening the endwalls reduced the crest loss coefficient and caused the coefficients to agree reasonably well with the longer overhang data and the curve. It is possible that flow separation around the short endwall reduced the effective crest length and thereby caused the increase in the crest loss coefficient. This effect would be relatively more important for the short dron inlets

The tests show that there is no effect of the antivortex plate overhang on the crost loss coefficient if long endwalls are used or if the overhang is 0.750 or greater.

Effect of drop inlet crest thickness.—The thickness of the drop inlet creat t, had a surpris-

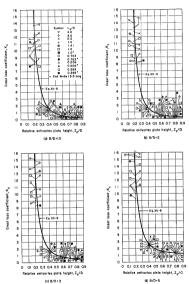
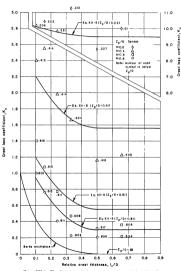


Figura XII-5.—The effect of antivories plate height and overhang on the crest loss coefficient, $t_i/0 \, \equiv \, 0.50$.



Fecuse XII-6.—The effect of crest thickness on the crest loss coefficient, $8/D \approx 2$.

ingly large effect on the crest loss coefficient for the thinner crests. This large crest loss coefficient for the thinner crest is possibly due to separation of the flow lines from the crest. For the thicker crests, the flow lines apparently follow the shape of the crest and the crest loss coefficients are smaller and constant. Hebaus's shows, however, that flow separation will occur for all crest thick-

nesses when $Z_i/D < \infty$. Six thicknesses of creats ranging from 0.1D to 0.860 were tested. Each creat was rounded on its more edge to a radius of hard the creat thickness inner edge to a radius of hard the creat thickness square. Four different heights of the antivorter square. Four different heights of the antivorter plate were used for each erset thickness. Typical results of the tests are shown in figure XIII-6. The curves shown inidicate the rangid decrease in the curves always inside the test of the curves are the constant of the coefficient tools increases for the smaller creet thickness, and the essentially constant value of the coefficient

for greater creat thicknesses. Regulation for the creat loss coefficient.—An equation was developed to represent the creat loss coefficient as a function of the relative thickness coefficient as a function of the relative thickness relative thickness of the relative thickness of the relative thickness of the relative thickness of the relative tempth of the drop link. This was accomplished in two steps. First, the creat thickness data shown typically in figure MILT-6 were reduced to a simple curve for each drop inlet length. Then an equation was derive for each drop inlet length. Then an equation was deep as a research of the following exhibitions are presented in the following exhibitions.

Reduction of creat thickness effect—In analysing the effect of creat thickness it was assumed, for thin creats, that the flow separates from the creat at the outside corners and forms a jet in the drop inlet allustrating this assumption is shown in figure XII-7. The contracted jet is twodimensional. Its thickness is D, feet and its vevelocity is V, feet per second.

If the head loss at the crest is due to the sudden expansion of the contracted jet to the full width of the drop inlet, then the crest head loss h, in feet can be obtained by rearranging an equation given by Rouse [I-44, p. 413, Eq. 41] to express the loss in terms of the lesser drop inlet velocity V, the drop inlet width D, and the jet thickness D. The resulting equation is

$$h_t = \left(\frac{D}{D_t} - 1\right)^t \frac{V_t^t}{\theta \cdot \alpha} \tag{XII-4}$$

If equation XII-2 is solved for h, and equated to equation XII-4, then the crest loss coefficient equation becomes

$$K_s = \left(\frac{D}{D_s} - 1\right)^2$$
(XII-5)

Because the flow is assumed to separate from the outside comer of the crest, the effective width of the entrance is $(D+2_2)$. This is shown in figure XII-7. When this width is multiplied by a suitable contraction coefficient C_s , the contracted width of the jet is obtained. When C_s , $(D+2z_c)$ is substituted for D_s in equation XII-5.

$$K_s = \left[\frac{D}{C_s (D + 2k_s)} - 1\right]' = \left[\frac{1}{C_s (1 + 2k_s/D)} - 1\right]'$$
 (XII-6)

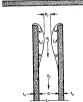


Figure XII-7.—Transverse section through top of drop inlet showing flow separation.

^{*}See page 929, figure 4, of reference in footnote 7. For the boundary flow line to restatch to the creat, the contraction coefficient must be 0.5 or greater. But figure 4 shows that the contraction coefficient approaches 0.5 asymptotically from below as the antivortex plate height

^{&#}x27;Sea bibliography for Part I, in Preface.

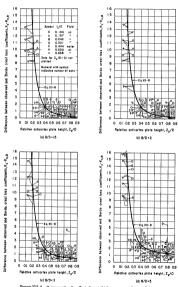


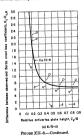
Figure XII-8.—Compensation for effect of creet thickness on creet loss coefficient.

This indicates that when 1/D > 0.5 the flow follows the wall of the drop intel, there is no jet expansion loss, and equation XII-6 is no inspeted on the property of the property of the prolate of the property of the property of the proservations apply only for a Borda monthplets, one of the property of the property of the proservations apply only for a Borda monthplets, above the creat that it has no effect on the creat below the creat that it has no effect on the creat loss conflictent. The observations do, however, indicate that the trand of equation XII-6 is similar to that shown by the experimental data presented to that shown by the experimental data presented by other [1-21, 1-22, and 1-24].

Strictly, the value of the crest loss coefficient for the Borda mouthpiece should be based on the velocity in a tube having a width of (D + 2k_s). The relation between the Borda tube velocity V_s, and the drop inlet velocity V, is

$$V_r \approx \frac{D+2\,t_0}{D}\,V_{r,\,0}$$

If this relationship is substituted into equation XII-4, if operations similar to those used to obtain equation XII-6 are performed, and if C_c is assigned the Borda mouthpiece value of 0.6, them the Borda mouthpiece creat loss coefficient K_a, is



 $K_{t,t} = \left(1 - 2\frac{h}{\pi}\right)^t$ (XII-7)

By trial a single curve was obtained for each top intellet length when the differences between equation XII-2 and equation XII-7 values of the creat loss oscillential was platford against the relative nutivorter plate height. The data care however in green XII-2 was a substantial to the control of the con

for the longer drop inlets. The curves of figure XII-8 give the increases in the crest loss coefficient $(K_c - K_a)$ that results from the decrease in drop inlet width attributable to the sidewall thickness as a function of the relative autivortex plate height. An advantage of these curves is that they reduce all the crest loss coefficients for different wall thicknesses to a common curve for each drop inlet length.

Development of the equation.—An equation representing the data in figure XII-8 was developed by:

oped by:

(1) Fitting empirical curves to the plotted data
for each drop inlet length.

(2) Reading points from the curves drawn to represent the data, and plotting the points on logarithmic paper.

(3) Selecting by trial a constant to subtract from each logarithmic curve to produce a single straight line on logarithmic paper.
(4) Plotting the subtractive constants against the relative drop inlet length on logarithmic

Traviers a straight line on this latter pide to present the suitherdier factor. The equation of the small line plotted on logarithmic paper to represent the direct of antivorse plate beight is 0.1(4,D)^{1.} The equation of the straight line plotted on logarithmic paper which represents the subtractive factor required to account for the effect of drop intel length is 0.03(4,D)^{10.} Therefore, the equation representing the data plotted in figure XII.-3 in figure XII.-3 in figure XII.-3 in

$$K_s - K_{s,\,s} = \frac{0.1}{(Z_s/D)^2} + 0.00 \left(\frac{B}{D}\right)^{1/3}$$
 (XII-8)

The curves drawn in figure XII-8 were computed using equation XII-8. They show that equation XII-8 represents the data satisfactorily.

The equation for the crest lose coefficient can be obtained by substituting equation XII-7 in equation XII-8 and solving for K. Thus

$$K_{s} = \left\langle 1 - 2 \frac{L}{\Omega} \right\rangle^{2} + \frac{0.1}{17 \cdot I \Omega V} + 0.00 \left(\frac{B}{\Omega} \right)^{1/2} \quad (XII-$$

The first term in equation XII-9 represents the effect of the relative crest thickness on the crest lose coefficient. It is valid only for $t_i/D \le 0.5$; for values of $t_i/D \ge 0.5$, the jet clings to the side of the drop inlet and the first term of equation 1XII-9 is zero—the pointed brackets indicate that the first term in equation XIII-9 is zero for negative values. The second term represents the effect for values. The second term represents the effect for the pointed brackets of the pointed for the presents the effect of the pointed for the present is the effect of the pointed for the presents the effect of the pointed presents the effect of the pointed presents and the present in the effect of the present in the effect of the present in the effect of the present in the effect of the present in the effect of the present in the effect of th

Precision of the equation.—The degree to which equation XII-9 represents the test data was determined by computing K, for each series for which the drop inlet dimensions meet the recommended criteria; that is, for all series for which equation XII-9 is valid. Computed values of K, and the percentage deviations from the observed values of K, are shown in table XII-1

The average error between the test data and equation XII-6 for the valid tests listed in tablable XII-1 is +22 percent. The maximum and mini-mum errors are +92 percent and -41 percent. The average error for the 23 valid water tests was +5 purcent while the average arror for the 252 valid air tests was +22 percent. However, the symond of the error, +92 to -41 percent for for the symond of the error, +92 to -41 percent for for the symond of the error, +92 to -41 percent for for the water tests and +98 to -22 percent for the stretch, was 40 excepts and +98 to -22 percent for the stretch, was 40 excepts and +98 to -22 percent for the water tests are 40 error.

No trand of the percentage error with respect to antivortee place begind or overhang could be detacted. There appears to be no trend of the detacted. There appears to be no trend of the percentage errors for rathiew drop intel lengths less than 50 but, the percentage error for 80–80 are also although the difference may be fortistive to the state of the percentage transplant for the rather than the colsered at the conditions of the percentage XIII when of the creat loss conflictent is less than the observed value for the thinner corets and becomes greater than the test value for the thinner corets and becomes greater than the test value for the thinner corets and becomes greater than the test value for the thinner corets and becomes greater than the test value for the thinner corets and becomes greater than the cost value for the thinner corets and becomes greater than the cost value for the thinner corets and becomes greater than the cost value for the thinner corets and becomes greater than the cost value for the thinner corets and becomes greater than the cost value for the thinner corets and becomes greater than the cost value for the thinner corets and becomes greater than the cost value for the thinner contains an extra percentage consistent of the cost value of the thinner contains the cost value of the thinner contains the cost value of the thinner contains the cost value of the thinner contains the cost value of the thinner contains the cost value of the

thicker crests. No attempt was made to improve equation XII-9 to eliminate this trend.

The precision of equation XII-9 appears to below. All efforts, however, to improve its precision have failed.

Barrel entrance loss coefficient

The barrel entrance energy loss coefficient describes the small energy loss that occurs between the midheight of the drop inlet and the barrel entrance plus the loss caused by the barrel on-tunce. The actual bead lost h, is the difference between the energy head at the drop inlet piezometer described under the paragraph leading. Creat loss coefficient and the energy head measured by projecting the friction gradelins in the bearrel to the barrel entrance.

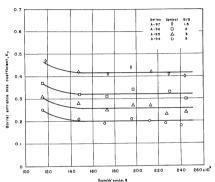
The barrel entrance loss coefficient K_1 is defined as $K_1 = \frac{h_1}{M \cdot 10^{-1}}$ (XII-10)

where $V_{+} = Q/A_{+}$ is the average velocity in the burrel in feet per second and A_{+} is the burrel area in square feet. The following peragraphs will disminate from the feet of the following peragraphs will distinct the feet of the feet of the feet of the feet burrel state of the feet of the feet of the feet of the feet ship appected, the burrel estimates of the feet of the feet ship appected, the burrel estimates of the feet of the state of the feet of the feet of the feet of the feet of the creat thickness or the amtivortex plate height or velapment of equations for the burrel entrance velapment of equations for the burrel entrance of the feet of the feet of the period of the feet of the feet of the feet of the feet of the original feet of the feet of the feet of the feet of the feet of the original feet of the feet of th

Hifted of Reynolds' number = The typical state of the Reynolds' number = V_0/V_0 on those barrel entrance loss coefficient is shown in figure XII-9. The curves drawn show the barrel entrance loss coefficient is constant for Reynolds' numbers exceeding 150,000. This in reasonables in agreement with the analysis of the results of tests on bends reported by Anderson'. This comparison is possible because the flow from the drop instet to the barrel is akin to that in a mitze bread, in the total parties is akin to that in a mitze bread.

The barrel entrance loss coefficients used in the subsequent analyses are the average of the experimental coefficients obtained for values of R ranging from a minimum of about 200,000 to the maximum obtainable with the air apparatus of

¹⁸ Anderson, A. G. Hydraulics of conduit bends. Univ. Minn., St. Anthony Falls Hydraul. Lab., Bul. No. 1, 22 pp., illus. December 1948, p. 9.



Proves XII-9.—The effect of Reynolds' number on the barrel entrance loss coefficient.

Growed-edged entrance, 5 = 0.4.

about 250,000. The maximum values of the experimental Reynolds' numbers approximate the minimum field values. Therefore, the applicability of the experimental results to field conditions assumes that the barrel entrance loss coefficient is constant for Reynolds' numbers exceeding about 200,000.

Effect of drop inlet length.—Figure XII-10 presents typical data showing that the barrel entance loss coefficient decreases as the drop inlet length increases. This is probably a result of the better approach conditions for the flow upstream of the barrel entrance which exist as the drop inlet is lengthened.

Effect of shape of barrel entrance.—The shapes of barrel entrance tested are shown in figure XII-1 and are described in the soction. Drop Inlet Description. A typical effect of the harrel entrance shape on the barrel entrance loss coefficient is shown in figure XII-10. To avoid contusion, only a sample of the data listed in table XII-1—that for $7_{\rm e}/D\approx0.8$ —has been plotted in figure XII-10.

The air data represented by open circles and squares show a small but possibly not significant increase in the square-edged barrel entrance loss coefficient if the crown of the barrel does not interact the downstream wall of the drop inlet. Newtrheless, separate curves are shown for each entrance. The water data, represented by solid circles, generally agree well with the open circle air data.

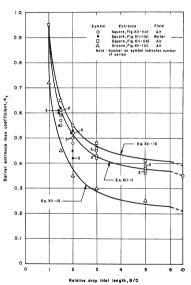


Figure XII-10.—The effect of drop inlet length on the barrel entrance loss coefficient, S = 0.2, $Z_c/D \approx 0.8$.

The groove-edged harvel entrance loss coefficient (represented by triangles) is considerably less than the square-edged coefficient. This size consists a direct measure of the expansion losses in the burrel, which are sensitive to the shaped of the burrel entrance edge. The larger barrel entrance resulting from the groove reduces the expansion loss as compared with the less for the square-edged burrel entrance.

Effect of harrel slope.—The barrel entrance loss coefficients, it decreases as the barrel slope increases. This effect is similar to that at ellows, and it is expected because the angle through which the water turns as it leaves the drop indet and enters the barrel decreases as the barrel slope increases. Typical data are shown in figure XII-ii.

As in figure XII-10, to avoid confusion only the data for Z₁O = 0.9 are plotted in figure XII-1.1 For the equire-deped entrance shown in figure XII-1.1 For the equire-deped entrance shown in the conjugate of the drop intel tength. To avoid overlap, the only data plotted for the square-deped enterpolar enterpolar district dept in the area of the conjugate of the conjugate of the prove-deped entrance shown in figure XII-1(c) are for the longest drop inlet. The detect of bernd lodge on the barrel entrance loss confidence is less for the inlet shown in figure XII-1(c) and the conjugate of the con

Equation for the barrel entrance loss coefficient.—The entrance loss coefficient for each barrel entrance is a function of the barrel slope and the dron julet length. The equation for this co-

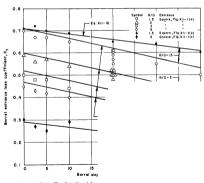


FIGURE XII-11.-The effect of barrel slope

efficient was developed from plots similar to that shown in figure XII-11, and its precision was checked against the original data. The findings are presented in the following subsections.

Descionant of the equation.—The data were free plotted as flow regirally in gang XII-11. On these plot is it was seen that the effect of hard rejee not be hard entired less called the effect of hard flow effect data way with the form of the hard shope effect data way with the form of the hard through the data for such hard extraors of the regreent the object effect. The intercept of these lines at zero hard slope were read from the plot. The intercepts are but represented by a constant that was a function of the hard entrance slape that the contract of the properties of the strength of the properties of the restriction of the hard entrance slape and the contract of the properties of the

The equations giving the barrel entrance loss coefficients K, for the three types of harrel entrances are:

For the square-edged barrel entrance shown in

$$K_1 = 0.43 + \frac{0.55}{16/0147} - 0.45 S$$
 (XII-11)

For the offset square-edged barrel entrance shown in figure XII-1(b).

figure XII-1(a).

$$K_1 = 0.43 + \frac{0.55}{19/D140} - 0.26 S$$
 (XII-12)

For the groove-edged barrel entrance shown in figure XII-1(c).

$$K_1 = 0.55 + \frac{0.55}{(B/D)^{5/5}} - 0.25 \, S \qquad (XII-13)$$

Although equations XII-11, XII-12, and XII-13 may not give a good estimate of the barrel entrance loss coefficient in individual cases, the average estimate of K, is good.

Entrance loss coefficient

The entrance loss coefficient represents the loss of energy between the headpool surface and the barrel entrance plus the loss in the barrel caused by the barrel entrance. An equation giving the entrance loss coefficient will be developed, and the procision with which the equation represents the data will be evaluated.

Development of the equation.—The entrance loss coefficient K, is

$$K_{\bullet} = \frac{h_{\bullet}}{V_p^2/2 g}$$
 (XII-14)

where V_p is the velocity in the barrel and $h_t = h_t + h_t \qquad (XII-15)$

The losses
$$h_c$$
 and h_t are defined by equations XII-2 and XII-10. When equations XII-14, XII-2, and XII-10 are solved for h_o , h_c and h_t these values are substituted in equation XII-15,

$$K_{\bullet} = K_{\circ} \left(\frac{V_{\bullet}}{V_{\bullet}}\right)^{2} + K_{\circ}$$
 (XII-16)

From continuity, A,V, = A_pV_p where A, = 8D is the area of the two-way drop inlet and A_p = π D'/4 is the area of the circular barrel. When these values are substituted in equation XII-16, the expression for K, becomes

and both sides of the equation are divided by the

velocity head in the barrel Va2/2 g.

$$K_1 = K_1 \left(\frac{\pi}{1-2m}\right)^2 + K_1 \quad (XII-17)$$

Presision of the equation.—The entrance loss conficient K, can be calculated from equation XII-17. Values of K, are obtained from equation XII-17. Values of K, are obtained from equation XII-11, XII-12, and XII-13. The computed values of K, for an experiment of K, and the percentage deviations from the observed values of K, are shown in table XII-1 for all series for which the drop indet dimensions meet the recommended criteria; that is, for all series for which causion XII-17 is with the configuration of the commended criteria; that is, for all series for which causion XII-17 is with the commended criteria; that is, for all series for which causion XII-17 is with the commended criteria; that is, for all series for which causion XII-17 is with the commended criteria; that is, for all series for which causion XII-17 is with the causion XII-17 is with the causion XII-17 is with the causion XII-17 is with the causion XII-17 is with the causion XII-17 is with the causion XII-17 is with the causion XII-17 is with the causion XII-18 is w

The average error between the test data and equation XII-17 for the 254 valid tests listed in table XII-1 is -3 percent. The maximum and minimum errors are +14 percent and -19 per-

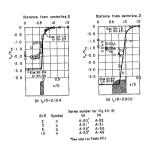


Figure XII-12.—Pressure distribution on the bottom of the antivortex plate, $Z_{s}/D = 0.1$.

cent. The average, maximum and minimum percentage difference for the 110 water series are -9, +13 and -19 and for the 144 air series are +1, +14 and -11. There are no readily apparent trends in the errors with respect to the drop intel geometry. The only trend noted is that the water differences are predominantly negative and greater in absolute value whereas the air differences are predominantly positive and smaller in absolute value whereas the air dif-

Although the prediction equations indicate maximum differences of +14 percent and -19 percent from the observed values of the entrance loss coefficients, the average difference is only -3 percent.

CAUTION regarding use of K_c and K_l to compute K_s .—Data for K_c and K_l obtained from dissimilar models cannot be used to compute K_s .

For example, the drop inlet piezometer tap in the air apparatus used to develop the equations for K, and K, was located on the drop inlet sidewall 1D upstream from the downstream endwall. For the water apparatus, the piezometer was located at the midlength of the drop inlet, which gives geometric similarity with the air apparatus only for the water drop inlet 20 long. The effect of this lack of similarity in the piecemeter locations in that the developed equations give values of K, for the long water drop inlets that are frequently more than 200 percent of the observed

values. Another example may be taken from the tests performed at the ARS Stillwater, Okla., Outdoor Hydraulic Laboratory. There the values of K, are computed using six plezometers at various locations in the drop inside, the Stillwater plezometer locations are not similar to those used as St. Anthony Falls and No. 1997. The Anthony Falls and No. 2007. The No. 2007 of the No. 2007

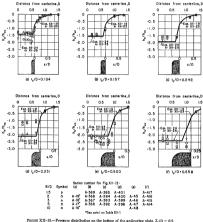
Valid and identical values of K, can be obtained only if the Stillwater and St. Authony Falls values of K, and K, are not combined to compute K, Conversely, valid and identical values of K, can be obtained if either, but not both, the Stillwater or St. Authony Falls values of K, and K, sreused to compute K.

Spillway Pressures

To structurally design the drop inlet, the pressures on the antivortey plate, the sidewalls and the endwalls must be known. Also, to determine if cavitation may occur, the pressures on the drop inlet crest and in the barrel just inside its entranca must be known

Pressures within the two-way drop inlet spillway were measured on the underside of the antivortex plate, on the drop inlet sidewalls (on the outside, the crest, and the inside), on the inside of the downstream drop inlet endwall and antivortex plate support walls, and in the barrel just downstream from the barrel entrance

Pressure coefficients are presented for use by designers. These pressure coefficients-when multinlied by the velocity head between the anti-



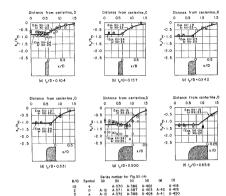


FIGURE XII-14.—Pressure distribution on the bottom of the antivortex plate, Z₀/D = 0.4.

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vortex plate and the crest, in the drop inlet, or in the barrel, whichever is applicable—give the actual pressure at the desired point.

Antivortex Plate Pressures

On the premise that the pressures on the underside of the antivortex plate are most closely related to the velocity between the antivortex plate and the drop inlet crest, these pressures were divided by the average velocity head between the antivortex plate and the drop inlet crest. This gave the pressures head coefficient for the antivortex plate at the drop inlet crest.

 $\frac{k_a}{k_{cr}} = \frac{\Delta p_a/w}{V_c^2/2 n}$ (XII-18)

where h, is the difference in head in fost, and p_a is the difference in pressure in pounds per square foot between any point a in the drop inled and the static head or static pressure couldn't be and the static head or static pressure couldn't be the company of the country of the country of the the everage velocity head between the antivortex plate and the drop inled creat in feet per secondi, vi is the specific weight creat in feet per secondi, vi is the specific weight creat in feet per secondi, vi is the specific weight creat infect per second, vi is the specific weight creat in feet per secondi, vi feet per second per second.

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Pressure heads on the underside of the antivortex plate are always less than the static pressure head at the same elevation. As a result, the pressure head coefficients are always negative.

The pressure on the underside of the antivortex plate was reasonable as one of the control of th

and 0.658 and drop inlet lengths of 1.5D, 2D, 3D and 5D.

The pressures on the underside of the antivortex plate are shown in figures XII-12 through XII-16. The pressure coefficients are negative; they add to the load on the antivortex plate.

The antivortex plate pressures outside the drop inlet will be analyzed separately from those inside the drop inlet.

Pressure outside the drop inlet

Subfigures (a) of figures XII-12 through XII-16, which present the results of tests on the crests 0.1040 thick where there were 15 piezo-

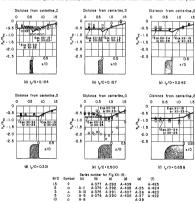


FIGURE XII-15.-Pressure distribution on the bottom of the antivortex plate, Z_s/D = 0.6.

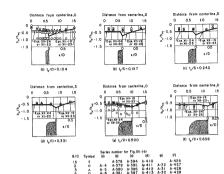


Figure XII-16.—Presence distribution on the bottom of the autivortex plate, $Z_{\nu}/0 = 0.8$.

meters on the antivortex plate, were used in the reeliminary analysis because these data best define the relationship between the pressure coefficients and the distance from the drop inlet. For each antivortex plate height an overlay curve was sketched and compared with the observed data. When a satisfactory fit had been obtained, an equation was developed that fit all of the overlay curves. This equation is

$$\frac{h_s}{h_{cr}} = -\log_{10}^{-1} \left[-0.125 - \frac{0.44}{Z_s/0} \frac{x}{D} \right] \quad (XII-19)$$

where x is measured outward from the outside well of the drop inlet.

Equation XII-19 has been plotted on figures XII-12 through XII-16. The equation gives a reasonable representation of the pressure on the underside of the antivortex plate. Appreciable deviation between the equation and the observations occurs only for the 10D-long drop inlets. Rouation XII-19 can be written using V. and h. the average velocity and velocity head in the riser or drop inlet respectively. Since Q

A-29

= V. 28 Z. = V. BD.

$$V_c = \frac{V_c}{4.7 \cdot / D} \qquad (XII-20)$$

(XII-22)

and
$$h_{ss} = \frac{V_{s}^{2}}{2g} = h_{ss} \frac{1}{[2\mathbb{Z}_{s}^{2}/0]^{2}} \qquad (XII-21)$$

$$\frac{h_{ss}}{h} = \frac{h_{ss}}{h} \left(2\frac{\mathbb{Z}_{ss}^{2}}{h^{2}}\right)^{2} \qquad (XII-22)$$

Substituting this value in equation XII-19, the pressure on the underside of the antivortex plate outside the drop inlet is, in terms of the velocity head in the drop inlet.

$$\frac{h_1}{h_2} = -\frac{1}{(2Z/D)^2} \log e^{-1} \left[-0.125 - \frac{0.44}{Z/D} \frac{x}{0} \right]$$
 (XII-23)

Pressure inside the drop inlet

An examination of figures XII-12 through XII-16 shows that the pressures on the underside of the antivortex plate are approximately constant across the width of the drop inlet.

To make the analysis, the average minimum pressure for all values of 8/0 was read from each subfigure of figures XII-12 through XII-16 and was tabulated. Manipulation showed that an approximate sureleope curve could be drawn on a plot of $(k_1k_2),(Z_p/D)$ versus k_1/D . The envelope curve of minimum average pressure on the underside of the antivortex plate within the drop inlet has the counsile

$$\frac{h_a}{h_m} = -\frac{0.85 + 0.95 \log_{10}(h/D)}{7_c/D}$$
 (XII-24)

Equation XII-24 has been plotted on figures XII-12 through XII-16. As aboven in figure XII-12, for the lowest antivortex plate the equation pressures are appreciably lower than the observed pressures. And, as shown in the (a) subfigures of figures XII-14 to XII-16, the computed pressures are higher than the observed pressures for the UD-long frop inlets. Otherwise the values of the pressures in the drop inlet computed pressures for the XII-14 to XII-16 to the values of the pressures in the drop inlet computed from equation XII-24 spaper to appreash

the data reasonably well.

Equation XII-24 can be written in terms of the drop inlet velocity head by substituting in it equation XII-22. Thus

$$\frac{k_h}{k} = -\frac{0.55 + 0.25 \log_{10}(k_b/D)}{4.77 \cdot (0.1)}$$
 (XII-25)

Pressure over the creat

The pressures on the underside of the antivortex plate and over the creat can be approximated by extending the pressures given by equations XII-26 or XII-26 to over the midpoint of the creat, then varying the pressure uniformly to the pressure given by equations XII-19 and XII-23 at XID = 0. the outler edge of the creat

The suggested approximation of the pressures over the crest is shown in figures XII-12 through XII-12 KII-16. The best evaluation of the approximation can be made in figure XII-12(b) and the (c) subfigure of figures XII-13 through XII-16. This is because the creat is relatively thick $(t_1/D=0.500)$ and more piezometers are available to make the evaluation.

Although the suggested approximation is a reasonable representation of the pressures on that

part of the underside of the antivortex plate which lies above the sidewall creat, it is likely that using the pressures given by equations XII-24 or XII-25 over both the inside of the drop inlet and the creats will be satisfactory for ordinary design use.

Comments

Equations XII-19, XII-23, XII-24 and XII-25 give the average maximum hydraulic loading on the antivortex plate for full conduit flow. Loadings for less than full flow will not exceed the full-flow loading.

If trash racks are used, these hydraulic losds apply only when there is a negligible head loss through the trash racks. The load resulting from partial or complete plugging of the trash racks must be added to the hydraulic losds presented herein.

Crest Pressure

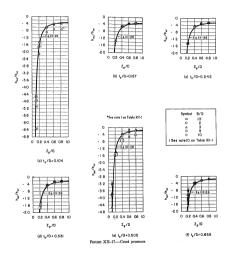
To evaluate the pressure on the drop linels cross, the pressure was measured at from two to four points. Pressures were always measured at the creat centerfine and at 45 degrees on the curved downstream naries of the creat. Pressures were also measured at the end of the creat curve were also measured at the end of the creat curve were also measured at the end of the creat curve were also measured at the end of the creat curve and notice of the curve and notice of the creat curve and notice of the curve and notice of the curve and notice of the curve and notice of the curve and notice of the curve and notice of the curve and notice of the curve and notice of the curve and notice of the curve and notice of the curve a

The pressure coefficients h_{α}/h_{α} are plotted in figure XII-17. All values of h_{α}/h_{α} are negative. Many of the pressures for $\theta/D=10$, which are shown as hexagons in figures XII-17(a) and XII-17(a) plot below the remainder of the data. The reason for this is unknown. These data were ignored in the subsequent analysis.

The axient of the vertical line through some of the data points represents the range of h_w/h_w observed at the several plezometer taps. In general, the smaller the symbol, the less the range of the observed pressures. Many of the pressures at the several taps on each creef agreed closely. For other creats the pressures at the several taps were significantly different. No regular pattern of the difference was discovered.

A hyperbolic equation was fitted to the data shown in figure XII-17. The equation is

$$\frac{h_m}{h_m} = \left\{4.5 \frac{h}{D} - 4.95 \right\} + \frac{1}{7.70} \left[2 - \frac{1.6}{(7.70)^{3/2}}\right]$$
 (XII-28)



where $\left\{4.5 \frac{h}{0} - 4.25\right\} \le -2$; that is, for $h/D \ge 0.5$ the quantity in braces is -2.

Equation XII-26 has been plotted in figure XII-17 to show that the equation well represents the data. Equation XII-26 has also been plotted in figures XII-19 through XII-23, which show the pressure distribution on the drop inlet

sidewall. In figures XII-19 through XII-23, equation XII-26 has been considered to apply along the crest curve.

Sidewall Pressure

The pressure on the drop inlet sidewall was measured by a row of piezometers located one pipe diameter upstream from the downstream end of the drop inlet. Pressures were measured on both the inside and outside of the sidewalls. The inside piezometers extended from the crest to the midbeight of the 50-bigh dwn inlet used for these tests. They were located at distances below the drop inlet crost r/D of 0.25, 0.50, 0.75, 1.00, 1.5. 2.0 and 2.5. An exception was the uppermost piezometer when L/D = 0.658, which was located at z/D = 0.329-the downstream end of the crest radius. The vertical distance below the drop inlet creat z is positive below the creat elevation and negative above the crest elevation. The outside piezometers extended only to 0.75D below the crest and were installed only on the dron inlets having wall thicknesses t/D of 0.104 and 0.500. These piezometers were located at z/D = 0.13. 0.25, 0.50 and 0.75

Pressure heads on the sidewall are always less than the static pressure head at the same elevation. As a result, the pressure head coefficients are always negative.

The pressures outside and inside the drop inlet will be analyzed separately, and then combined to determine the net load on the sidewall.

Pressure inside the drop inlet

The pressures near the top of the inside side. wall of the drop inlet were initially analyzed on the premise that the stream would senerate from the sidewall, at least for the thinner sidewalls, and that the sidewall pressure in the separation zone would be constant and equal to the pressure on the sidewall crest given by equation XII-26. On the other hand, the sidewall pressures near the midheight of the drop inlet were initially analyzed on the premise that the flow was essentially uniform, that the pressure coefficient at the midheight would be - (Ke + 1) where Ke is computed from equation XII-9, and that presaures above the midbeight would be a function of the pressure gradient. The problem, then, was to determine the regions where each premise was valid: to determine and evaluate the effect of antivortex plate height, sidewall thickness and drop inlet length on the sidewall pressures; and to devise methods for evaluating the pressures necessary for the structural design of the drop inlet sidewall.

An initial step in the analysis was to divide the relative pressure h_i/h_{ν_i} measured at each pressure tap on the drop inlet sidewall by: (1) the relative pressure on the crest h_{ν_i}/h_{ν_i} computed using equation XII-26, and (2) the quantity - ($K_c + 1$) with K_c computed using equation XII-9.

The first ratio, $\frac{(h_n/h_n)}{(h_{nc}/h_n)}$ computed, should be 1.00

if the observation point is in the separation zone. Similarly, the second ratio,

(h_o/h_o) observed , should be 1.00 at the midheight of the drop inlet and should deviate slightly from 1.00 at points above the midheight if the flow in the drop inlet is uniform. These are the principal ratios, based on the previously men-

tioned premises, used in the subsequent analysis. The two ratios computed for each series were tabulated side by side to facilitate comparison. The ratio closest to 1.00 determined which denominator used to normalize the data best represented the experimental results. The number of series, expressed as a percentage, in which the ratio based on $-(K_* + 1)$ is closer to 1.00 than the ratio based on the creat pressure (equation XII-26) is listed in table XII-4 for each piezometer and crest thickness Percentages exceeding 50 indicate that - (K. + 1) is the best normalizer; percentages less than 50 indicate that the crest pressure is the best normalizer, that is, that the piezometer is located in or is primarily affected by the separation zone. The tabulated percentages indicate that the creat pressure best normalizes the observed pressure for the pressure tan 0.25D below the drop inlet creat (*/D -- 0.25) and for crest thicknesses less than 0.500D when z/D = 0.50 and crest thicknesses less than 0.331D when z/D = 0.75. With one exception, the pressure at the drop inlet midheight. — $(K_* + 1)$. best normalizes all other observed pressures.

Plotting the best normalized ratios averaged for all variables except antivortex plate height

Table XII-4.—Percentages of inside sidewall pressure observations best normalized by — (K_t + 1)

10				1/D			
Ď	0.25	0.60	0.75	1.03	1.50	2.00	2,50
0.104	40	35	38	57	85	95	100
.167	0	0	12	24	62	82	100
.242	12	25	44	59	71	94	100
.331	25	44	62	69	88	100	100
.600	25	60	60	65	70	85	100
.658	22	53	71	75	81	94	100

against the antivortex plate height showed only random variation. As a result, no further correction for the effect of the antivortex plate

height was necessary. Plotting the best normalized ratios averaged for all variables except drop inlet length against drop inlet length showed a systematic variation for all piezometers except that piezometer located at the drop inlet midheight. Because preliminary analyses and considerations based on analysis of streamlines plotted with the aid of conductive paper indicated that the measured pressures are a function of the drop inlet length, this variation was not unexpected. However, when z/D = 0.50 and 0.75, a more orderly arrangement of the variations between the several piezometers was obtsined when - (K, + 1) was used to normalize the pressures for all crest thicknesses instead of only those piezometers where the percentage in table XII-4 exceeds 50. As a result, the crest pressure was used to normalize the pressure only

A means was sought of describing the variation of drop inlet sidewall pressure with drop inlet length. For the midheight piezometer (x/0=2.5), the drop inlet length effect is included in the computation for K, and no further correction for drop inlet length is required. This is demonstrated in figure XII-18(0,0) where the value of

for the piezometer located at r/D = 0.25.

the ratio
$$\frac{(h_n/h_{sr})}{-(K_s+1)} \frac{\text{observed}}{\text{computed}}$$
 is close to 1.00

and is constant with respect to the relative drops intel length \$P.O. Such constancy, however, does not exist for the piezometers located above the drop intel midheight. This is demonstrated in figure XIII—18(b), (c), and (d), where the variation of the ratio with drop intel length increases with the distance from the midheight piezometer (decreasing r/D).

When the logarithm of the ratio $(h_c/h_{sc}) / [-(K_c + 1)]$ —the ratio of the ob-

served pressure head to the velocity head in the drop inlet normalized by the pressure coefficient at the drop inlet midheight—was plotted against (8/D):—the square of the relative drop inlet length—a systematic trend was apparent. This trend can be represented by the equation

$$K_{bo} = \frac{1.742 - \log_{10} |x/D|}{1.344} \ _{b} - (8/D)^{1} \quad (XII-27)$$
 for $8/D \le 7$ (for $8/D > 7$, use $8/D = 7$) where

wnere



Figure XII-18:—Sidewall pressure divided by computed midbeight pressure, Z₂/D = 0.8.

 K_{N0} is the multiplier correction for drop inlet length to be applied to $-(K_x+1)$, and for

a/D = 0.5, 0.75, 1.0, 1.5, 2.0, 2.5, b = 1.0145, 1.0125, 1.0112, 1.0090, 1.0047, 1.0010,

(For $0.5 \le z/D \le 1.5$.

$$b = 0.9925 + 0.0145 \left[\frac{1.742 - \log_{10} (x/D)}{1.244} \right],$$

To test equation XII-27, the ratio $(k_h/k_{\rm p})$, $[-k_{\rm p} (k_{\rm r}+1)]$ was computed for each test series. When this ratio, averaged for each crest thickness i/D, was plotted against i/D a systematic trend was apparent. This indicated a need to apply a correction for the crest thickness effect.

The multiplier correction for the effect of crest thickness K₁₀ is represented by the equation

$$K_b/D = \{0.66 + 0.06 \frac{\pi}{D}\} \left(\frac{L}{D}\right)^{-\frac{1}{2}} \left(0.276 - 0.079 \frac{\pi}{D}\right)$$
 (XII-28)

where $0.5 \le x/D \le 2.5$. To test the product of equations XII-9. XII-27, and XII-28 against the observed data, the ratio (h_r/h_r) / $[-K_{rm}K_{rm} (K_r + 1)]$ was computed for each test series. Compared with the ratios computed using the Kup correction only, there was a considerable improvement in the agreement of the equations with the original data averaged for each crest thickness. The average ratios for all tests showed little change, but there was a reduction in the probable error as a result of correcting for the crest thickness effect. The average ratio and the probable error am shown in table XII-5. Excluded from these figures are the data for B/D == 10, which were inconsistent with the remainder of the data, and the data for Z_/D \approx 0.1, which are for an antivortex plate height lower than is recommended.

A column for z/D = 0.25 was added to table XII-5. The values given by equation XII-26 multiplied by 0.93 were used to normalize the pressures for this niezometer.

The agreement of the equation with the observed sidewall pressures is shown in figures XII-19 through XII-23 for all available pertinent data. It can be een that the equations do not represent the pressures when 8/D=10, this observation agreeing with the previous statements that the data for 8/D=10 are inconsistent with the remainder of the data. In addition, the equations are not representative of the data shown in figure XII-19 where X/D=0.1; for

Table XII-5.—Average agreement of observed inside sidewall pressures with computed

	1/0						
Item	0.25	0.60	0.75	1.00	1.50	2.00	2.50
Ratio	1.00	.99	.99	.98	.97	.98	.99
error	.08	.11	.11	.11	.09	.07	.05

 $\mathbb{Z}_q/\mathbb{D}=0.1$ the equation values of \mathbb{N}_p , range from -91 to -181 when $\mathbb{N}_q^2 > 0.104$ whereas the data are in the range -44 to -5, and when the data are in the range -44 to -5, and when $\mathbb{N}_q^2 > 0.000$ the values of $\mathbb{N}_q^2 > 0.000$ the range from -66 or -30. Furthermore, for $\mathbb{Z}_q/\mathbb{D}=0.2$, figure XII-20, the equations poorly represent the data. However, because "X" in the notes of figures XII-40 in and XII-20 means that the "drop intel dimensions of not meet the recommended criteria, the served that is of to nucleoid similarities the exceeded that is of to nucleoid similarities.

The equations do well represent the observed date, except for \$D_i = 10 as noted above, when \$A_iD_i\$ is \$D_i\$ or \$D_i\$





Figure XII-19.—Pressure distribution on the drop inlot sidewall, $Z_s/0 = 0.1$.

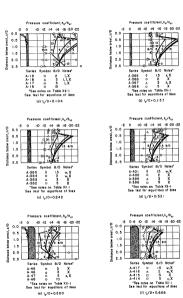
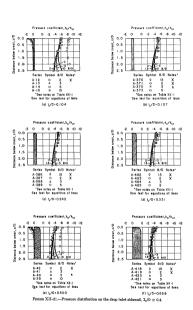
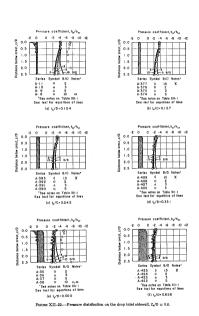
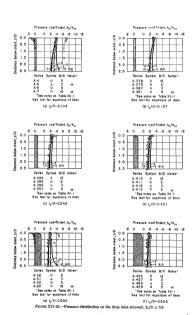


Figura XII-20.—Pressure distribution on the drop inlet sidewall, $Z_p/D = 0.2$.







Summary.-To compute the sidewall pressures inside the drop inlet:

when z/D = 0.25, multiply the value obtained from equation XII-26 by 0.93:

when $0.5 \le x/D \le 2.5$, multiply — $(K_* + 1)$, with K, computed by equation XII-9, by the drop inlet length correction given by equation XII-27 and by the crest thickness correction

given by equation XII-28. The average precision of the prediction is indicated in table XII-5. The agreement of the equations with the observations is shown in figures

XII-19 through XII-23. Pressure outside the drop inlet

and the creat her.

Sidewall pressures outside the drop inlet were measured at distances below the creat z/D of 0.13. 0.25, 0.50, and 0.75 for drop inlet crest thicknesses t./D of 0.104 and 0.500 and drop inlet lengths B/D of 2 3 5 and 10 It was reasoned that the pressures h, would be most closely related to the velocity between the antivortex plate

An inspection of the tabulated values of h./h., indicated that, with respect to the distance below the drop inlet crest z/D, the relationship was hyperbolic, A tabulation of the product (ha/ha) (z/D) showed the product to be substantially constant for each antivortex plate height Z./D.

When (h./h...) (z/D) was plotted against Zn/D, the relationship was adequately represented by a straight line. The resulting equation for the relative pressure on the outside of the drop inlet sidewall is

$$\frac{h_n}{h_{10}} = -\frac{Z_0/D}{8\,x/D} \tag{XII-29}$$

which can be reduced to

$$\frac{h_b}{h} = -\frac{1}{2}\frac{Z_p}{h}$$
(XII-90)

(XII-30)

Using equation XII-21, equation XII-29 can be expressed in terms of the velocity head in the drop inlet. Thus

$$\frac{h_{o}}{h_{eff}} = -\frac{1}{32 (Z_{o}/D) (s/D)}$$
 (XII-81)

Equation XII-29 was tested by computing the differences between the computed and the observed relative pressures ha/hac. The maximum average error is 0.05 and the maximum probable error is 0.03. In general, the errors increased as the antivortex plate height increased. A reason for this may be that the velocity between the antivortex plate and the crest is low for high antivortex plates. When the sidewall pressure is divided by the low velocity head, any error in the pressure measurement is magnified, so there may be a larger error for the high antivortex plates than for the low antivortex plates where the velocities are higher. However, the precision of the results is well within practical needs.

Equation XII-31 has been plotted on figures XII-19 through XII-23. It is apparent that the equations well represent the data.

Net pressure on the sidewall

The net pressure load on the sidewell is the difference between the pressures inside and outside the drop inlet. When the drop inlet is completely full, the

pressures outside the drop inlet are computed using equation XII-31. The pressures inside the dron inlet are given by equation XII-26 over the crest curve, by 0.93 times the result obtained using equation XII-26 when 1/D = 0.25, and by $-K_{\text{trip}} K_{\text{trip}} (K_e + 1)$ when $0.5 \le z/D \le 2.5$ where Kup is given by equation XII-27, Kup is given by equation XII-28 and K, is given by equation XII-9. Because the inside pressures are always

less than the outside pressures, the net load is inword When the drop inlet is partly full, as is the case for weir flow over the drop inlet crest, the pressures inside the drop inlet are approximately atmospheric. The pressures outside the drop inlet are the static pressures which near the crest

and the crest thickness; and outside the drop inlet there were from 0 to 12 piezometers. Only for crest thickness t./D of 0.104 and 0.500 were there sufficient piezometers outside the drop inlet to permit sketching the complete pressure contours. When t/D = 0.104, the overhood of the endwalls L/D was 1.86; and for t/D = 0.500. L./D was 1.50. For other crest thicknesses, La/D varied from 4.0 to 3.5 depending on the crest thickness, the same endwall being used for all crest thicknesses. Despite the large number of piezometers, there were far too few to permit accurate drawing of the pressure contours, especially in the vicinity of the crest where the endwall pressures vary rapidly with distance in any direction

Pressure heads on the endwall are always less than the static pressure head at the same elevation. As a result, the pressure head coefficients h./h. are always negative.

All attempts to reduce the endwall pressures to a common denominator led to unastifactory presentations of the data. One of these attempts involved normalizing the pressure coefficients using the computed pressure at the drop inlet midheight – (K, + 1), K being computed from equation XII-9, Another attempt, equally unsuccessful, was to correct – (K, + 1) using K₀₀ and K_{1,00} computed from equations XII-27 and C_{1,100} computed from equations XII-27 and XII-28.

The onalytical procedure ultimately adopted was to first record the pressure coefficients for each test series on a drawing of the endwall elevation. To assist in extending the contours to the underside of the antivortex plate, the antivortex plate pressures measured 1D upstream from the downstream endwall were added to the work plots.

Pressure coefficient contours were then drawn. Inspection showed that the pressure coefficient contour patterns were approximately similar for each crest thickness, but that the magnitude of the pressure represented by each contour varied greatly with antivortex plate height and slightly with drop inlet length. Therefore, a typical pressure coefficient contour pattern was sketched on tracing paner for each crest thickness.

The contour pattern was then placed over the plotted data for each test series, and representative pressure coefficients for each contour were read and tabulated. A study of the tabulations for drop inlet lengths 8/0 of 1.5, 2, 3, and 5 demonstrates.

stated, within the limits of experimental precision and the variations in the sketched locations of the contours, that the variation with B/D was not significant. Therefore, a single tabulation of the pressure coefficient $-h_0/h_0$, for each contour was prepared for each crest thickness i_0/D and i_0/D with i_0/D is value of $-h_0/h_0$, then being a function of the contour and the arrivotex palse height J_0/D .

The experimental results are presented in figure XII-24 in the form of lettered pressure coefficient contour lines for each crest thickness tested. The value of $-h_{\nu}/h_{\nu}$ for each contour is given in an eccompanying table as a function of the antivortex plate heleful.

A quick check on the precision of the summatics presented in figure XII—2 indicates that they represent the observed pressures to within perhaps 25 percent on the average. As noted previously, the rate of change of pressure with distance in the vicinity of the drop intel creat is large, and there were too few piezometers to accurately define the pressure noticent there.

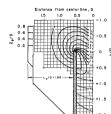
Barrel Entrance Pressure

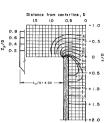
Low pressures exist just inside the entrance to the harrel. If these pressures are low enough, they can induce cavitation which may, in turn, damage the conduit. Information to permit evaluating these pressures will be presented for the three types of barrel entrances shown in the subfigures of figure XII-2.

The analysis was made in terms of the pressure coefficient for the spillway barrel. The pressure coefficient for any point n along the barrel is defined as h_i/h_{n_p} , where h_i is the deviation from the friction gradeline at n and h_{n_p} is the velocity head in the pipe.

Point pressures were measured at both the barrel crown and invert at a distance D/2 downstream from the barrel entrance. Pressures were also measured at 17D, 33D, 49D, 65D, 81D, 97D, 113D, 129D and 137D along the barrel.

The disturbance caused by the barrol entrance is almost completely damped cut at 170, so pressures at 350 and beyond were used to compute the friction gradeline. The least squares method was used for this purpose. The measured and least squares grade line pressures beyond 170 agreed to within ±5 prenent of the pipe velocity head. The least squares friction gradeline was protected back to the barrel entrance. The difference.





Values of - h_/h_vr

ontour	Z _p /D					
omoor	0.206	0.412	0.616	0.820		
A B C D	0.25 .5 .8 1.4	0.20 .3 .5 .7	0.15 .20 .3 .4	0.10 .18 .2 .3		
E F G H	3.0 6.0 9.0 13.0	1.2 2.0 3.0 4.0	1.0 2.0 2.9	.4 .7 1.5 2,2		
J K L	14.0 15.0 16.0 16.3	5.0 5.5 5.7 6.2	3.5 3.8 4.0 4.1	2.8 3.1 3.4 3.7		
M N O P	14.0 13.6 13.1 12.5	5.1 4.7 4.3 4.0	3.8 3.4 3.0 2,5	2.9 2.7 2.4 2.2		

Values of -h_/h_ve

Values or - n n n vr						
Contour	Z _p /D					
Comou	0.223	0.417	0.614	0.813		
B C D	0.6 .8 1.0 3.0	0.4 .5 .6 1.0	0.2 .3 .4 .6	0.15 .2 .3 .4		
E F G H	5.0 8.0 12.0 13.6	1.5 2.0 3.3 4.5	1.2 2.0 3.0	1.0 2.0 2.5		
J K L	13.9 14.0 13.7 13.4	5.0 5.1 5.0 4.6	3.4 3.5 3.1 2.8	2.8 3.0 2.5 2.2		
м	13.1	4.4	2.5	2.0		

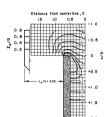
(b) t_c/D = 0.157

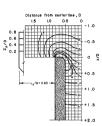
Figure XII-24,-Pressure distribution on the drop inlet downstream endwall.

ence between the hydraulic gradeline and the friction gradeline is h_{ν} . A more detailed explanation of h_{ν} its significance, and its use is given on

page 13 of Part I³¹ of this closed conduit spil¹way report series.

"See Part I reference in Preface for complete citation.





Volues of -hm/hvr

Contour	Z _p /D					
Comoor	0.222	0.414	0,613	0,813		
8 C D	0,4 .8 1.5 3.0	0.3 .5 .7 1.5	0.2 .3 .5	0.15 .2 .3 .5		
E G H	10.0 12.5 13.0 13.6	2.5 4.0 4.7 4.8	1.2 2.3 2.7 3.0	1.0 1.5 1.8 2.0		
J K	13.0 13.0 12.5	4.5 4.2 3.8	2.5 2.2 2.0	1.7 1.6 1.5		

Values of ~ h_m/h_{vr}

Contour	Z _p /D				
Confour	0,213	0,404	0.604	0.804	
A 8 C D	0.3 1.0 1.7 4.5	0.25 .6 .8 2.0	0.2 .3 .5	0,15 .2 .3 .5	
E F G H	15.0 15.0 15.2 15.0	3.0 4.0 4.3 4.4	1.5 1.8 2.0 2.2	1.0 1.2 1.3 1.4	
J K Ł	15.0 15.0 14.0 13.0	4.5 4.0 3.5 3.2	2.3 1.8 1.7	1.5 1.3 1.3	

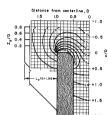
(c) to /D = 0.242

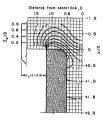
(d) ta/D = 0.331

FIGURE XII-24.—Continued.

For the square-edged entrance \underline{o} data were obtained for drop inlet lengths B/D of 1.5, 2, 3 and 5, for creat thicknesses L/D of 0.14, 0.17, 0.24, 0.331, 0.500 and 0.658; for antivortex plate heights Z_p/D of 0.14, 0.2, 0.4, 0.6, and 0.6; and for heights Z_p/D of 0.1, 0.2, 0.4, 0.6, and 0.8; and for herrol slopes S of 0.00, 0.025, 0.05, 0.10, 0.20, 0.30, 0.30

and 0.40. For the square-edged entrance \underline{b} similar ranges of Z_p/D and S were tested, but an additional drop intel length—B/D = 1—and only one crest thickness— $1_b/D = 0.500$ —were used. For the groove-edged entrance \underline{c} the ranges of B/D and $1_b/D$ were the same as for the square-edged





Values of - h_/h_

			**			
Contour	Z _p /D					
Confour	0.212	0,405	0,605	0.803		
A B C D	1.5	1.5	.4 :6 :7	0.2 .3 .4 .5		

Values of -h_h_

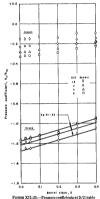
Contour	Z _p /D					
Comosi	0,220	0.416	0.610	0,808		
A B C D	0.30 .60 2.0 6.0	0.19 .30 .70 1.0	0.13 .20 .40 .5	0.10 .18 .25		
E F G H	8.0 11.0 12.0 13.0	1.5 2.0 2.5 2.9	.7 .8 1.3 1.4	.4 .5 .7 1.3		
J K L	13.2 13.4 13.4 12.4	3.0 3.5 3.8 2.6	1.5 2.0 2.6 1.4	1.3 1.5 1.5 1.3		
м	11.5	2.5	1.4	1.3		

(e) t_e/D = 0.500

FROUSE XII-2

entrance <u>b</u>, but only one antivortex plate height —Z_i/D = 0.8—was used.

Pressure coefficients are presented in figures XII-25, XII-26, and XII-27 for the three barrel



square-edged barrel entrance g. s./D = 0.500, Z_s/D = 0.80.

it is possible to compare the three entrances. From these figures it can be seen that most of the linest barrel entrance pressure coefficients are a stilked hadve the friction gradeline (L/h_b is regarded), and the control of the Because it is the lower crown pressures which are most likely to produce cavitation, no analysis of the invert pressures was attompted. The crown pressures, however, were analyzed to evaluate the effect on them of creat thickness, drop intellegith, and barrel slope. These analyses were made for seek between

Effect of creet thickness

Only for the square-edged entrance a were data available to determine the effect of crest thickness on the barrel entrance crown pressure.

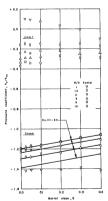


Figure XII-26.—Pressure coefficients at D/2 inside square-edged barrel entrance <u>b</u>. h/D = 0.500, Z. $D \approx 0.81$.

The analysis was made for one barrel slope— S = 0.2—the only slope for which complete data were available. Because there was no apparent systematic variation of the crown pressure coefficient with the crest thickness, the working plot is not presented here. There is, in fact, little reason to suspect that the crest thickness would affect the pressures in the harrel; the available data confirm this reasoning.

Effects of barrel slope and drop inlet length

The effects of barrel slope S on the pressure codicients are shown by the slopes of the lines in figures XII-28, XII-28 and XII-27 for the three types of barrel entrances tested. The effect of relative drop inlet length 8/D is indicated by the location of the lines. The straight lines shown in the figures can be defined by an equation having the several flows.

$$\frac{h_a}{h_{\phi}} = a + b S$$
 (XII-32)

where o, the intercept at S=0—different for each line—varies with the drop inlet length and b determines the magnitude of the barrel slope effect. The problem, then, is to evaluate α and b for each barrel entrance.

Equation for square-edged barrel entrance g.

—The lines drawn in figure XII-25 are parallel
so the coefficient b is constant for all drop inlet
lengths. It has a value of 0.55.

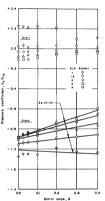
The intercept o of the line for each 8/D was read from figure XII-25 and an equation developed for o. It was found that the equation best representing o is o = $-1.865 + 0.5 \log_{10} (8/D)$ for values of 8/D between 1.6 and 2.4. The drop intel length had no effect on the barrel entrance crown pressure coefficient for values of 8/D exceeding 2.4; for drop intel lengths longer than 2.40, 8/D = 2.4 should be used in the equation for o.

The equation representing the pressure coefficient at the crown of square-edged barrel entrance g at a distance D/2 inside the barrel entrance is

$$\frac{h_b}{h_{10}} = -1.365 + 0.8 \log_{10} \frac{8}{D} + 0.865$$
 (XII-33)

for $1.5 \le B/D \le 2.4$. (For B/D > 2.4, use B/D = 2.4.)

The agreement of equation XII-33 with the dats is shown in figure XII-25. In addition, equation XII-33 was tested by computing values of



Facum XII-27.—Pressure coefficients at D/2 inside groove-edged barrel entrance g t_s/D = 0.500, Z/D = 0.812.

b_b_a and comparing them with the observed values. The compared values of k_b_w were lower than the observed values. The maximum deviation was 0.1 is, but only 16 of the 75 computed than 0.05 brrel velocity based. There was no detectable variation of the deviations with respect to crest thickness or barrel alope, but the difference did increas somewhat with respect to difference did increas somewhat with respect to the lower native value of the comparing the values for the lower native value. Secure the magnitude of the variation was small (about 0.04 h_r), no attempt was made to correct for the apparent slight effect of antivortex plate height.

Equation for square-edged barrel entrance b.—As in figure XII—25, purallel lines have been drawn in figure XII—25 to prepenent the crown pressure ocefficient D/2 inside the square-edged barrel entrance b. The slope of these lines—co-efficient b in equation XII—32—is 1/3, and constant to is -1.385 + 0.5 log₂₀ (8/D) for values of 8/D between 1,5 and 2.4.

The equation representing the pressure coefficient at the crown of the square-edged barrel entrance \underline{b} at a distance D/2 inside the barrel entrance is

$$\frac{h_s}{h_p} = -1.385 + 0.5 \log_H \frac{8}{D} + \frac{5}{3}$$
 (XII-34)
for $1.5 \le B/D \le 2.4$. (For $B/D > 2.4$, use $8/D = 2.4$.)

The agreement of equation XII-34 with the data is shown in figure XII-26. The agreement is excellent except for B/D = 1, a drop inlet length which does not meet the limiting design criteria. In addition, equation XII-34 was tested by comparing the computed and observed values of ha/ha. The agreement of the computed and observed values is excellent, averaging better than about 0.01 barrel velocity heads. Deleted from the average were the values obtained for drop inlets which do not meet the recommended criteria. The computed pressures were lower than the observed pressures, but the maximum difference of 35 observations was only 0.06 barrel velocity heads. There was no observable trand of the differences with either antivortex plate height or harrel slone.

Equation for groove-edged barrel entimone, -The lines drawn in figure NLT-9 to appresent the crown pressure coefficient D/2 inside groover coefficient D/2 inside groover coefficient between the pressure coefficient and the barrel solps. The dope of these lines—coefficient b in equation XII-32 s he 0.88 - 0.96 / (40)0 and constant a is -1.01 + 0.38 logs, (8/D) for values of 3/D between 1 and 2.3.

The equation representing the pressure coefficient at the crown of the groove-edged barrel entrance \underline{c} at a distance D/2 inside the barrel entrance is

$$\frac{h_s}{h_p} = -1.01 + 0.31 \log u \frac{8}{0} + (0.85 - \frac{0.94}{8/D}) S$$
 (XII-36)
for $1 \le B/D \le 2.2$ in the second (log) term. (For $B/D > 2.2$, use $B/D = 2.2$.) In the third (S) term $1 \le B/D \le 5$.

The agreement of equation XII-35 with the data is shown in figure XII-27. Even for B/D == 1. a drop inlet length shorter than recommended. the equation well represents the data. In addition, equation XII-35 was tested by comparing the computed and observed values of ha/has. The agreement of the computed and observed values is again excellent, averaging about 0.01 pipe velocity heads. Deleted from the average were the values obtained for drop inlets which do not meet the recommended criteria. The computed pressures were both greater and less than the observed pressures, the maximum difference of 28 observations being only +0.03 and -0.02 barrel velocity heads. Again, there was no observable trend of the differences with harrel slope. Because only one antivortex plate height was used for these tests, the effect of antivortex plate height. if any, could not be determined.

Recommendation Summary

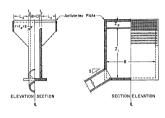
The only information included in this summary of the results of the two-way drop inlet spillway tests are the recommondations for proportioning the drop inlet, for determining the spillway capacity, and for evaluating the pressures. The bases for three recommendations can be found in the text.

For convenience of use, the recommendations are presented in:

Figure XII-28 for proportioning the drop inlet, Figure XII-29 for determining the spillway capacity, and

Figure XII-30 for evaluating the pressures in the spillway. With the single exception of the minimum antivortex plate height, all of the results in figures XII-28, XII-29 and XII-30 are expressed in dimensionless units or in a dimensionally correct form. Therefore, any consistent system of units—for example, English or metric—can be used. In dimensionless units, the criteria for minimum antivortex plate hight have for

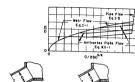
mum antivortex plate height are, for B/D = 1.6, $Q/\sqrt{5}$ $D^{1/2} \approx 1.8$ and, for $B/D \ge 2$, $Q/\sqrt{5}$ $D^{1/2} \approx 1.0$. To make equations 1-1 and 1-2 dimensionally correct, add the terms $\sqrt{2}$ to the right side, the values of the dimensionless discharge coefficient $C/\sqrt{2}$ 5 then being 0.39 for C = 3.1 and 0.50 for C = 4.0.



DROP INLET HEIGHT, Z. MAXIMUM ANTIVORTEX PLATE HEIGHT, Z. The headpool eurtoce elevation of which the weir flow and pipe flow discharges are equal—the interaction of the weir and pipe flow head-discharge oursee in, for example, Z, ≥ 30 DROP INLET LENGTH. B B ≥ 1,5 D Fig. XII-29. MINIMUM ANTIVORTEX PLATE OVERHANG, L. MINIMUM ANTIVORTEX PLATE HEIGHT, Z. is the wair flow head for Table XII-3. Minimum Antivortex Plate Overhang Q2 7.40** when B - 1.50 (L_/D)_{minimum} Q2 5.10 when B2 20 1.5 1.0 2 Toble XII-2. Minimum Antivortex Plate Height 5 0.4 (Z,/D). 0.1 C - 3. C- 4.0 0.85 0.72 * To eliminate effect of 0.55 0.47 entivertex plate everhong 0.30 0.25 0.30 0.16 SLDPE. S G is the weir flow coefficient in the equation 0.2 only slope tested Q • CL H *** (I-I)

PISURE XII-28.—Recommended proportions of the two-way drop inlet.

(I-2)



Entrance a

Entrance b



(XIII-III)

WEIR FLOW

FLOW Use published equations or Q=CLH^{B/E} (Eq.I-I) where L=28 and C is evaluated from published information. ANTIVORTEX PLATE FLOW *

$$\frac{H}{D} * \frac{Z_p}{D} = \left\langle \frac{Q_1 Z}{B/D} - Q_1 | \log_{10} \frac{L_0}{D} \right\rangle + \left\langle Q_1 | -Q_1 Q_2 \frac{L_0}{D} \right\rangle \frac{Q}{\exp(\frac{1}{2}Z_p)} \ge \frac{Z_p}{D} \tag{XII-1}$$

PIPE FLOW

$$Q = \frac{\pi}{4} D^{\frac{p}{4}} \sqrt{\frac{2gH_1}{K_0 + K_0 + \dots + f\frac{g}{D} + f_1 \frac{g_1}{4R_1} \left(\frac{A_D}{A_1}\right)^{\frac{p}{4}}}}$$
(1-5)

Crest Loss Coefficient*

$$K_0 = \left\langle 1 - 2 \frac{1}{D} \right\rangle^2 + \frac{Q_{,1}}{\left(Z_{,p}',0\right)^2} + Q_{,0}Q_{,p}^2 \left(\frac{B}{D}\right)^{9/2}$$
(XII-9)

Barrel Entrange Lage Coefficient Square - edged borrel entronce

Offset square-edged barrel entrance b

$$K_1 = 0.43 + \frac{0.55}{(8/0)^{8/6}} = 0.258$$
 (XH-12)

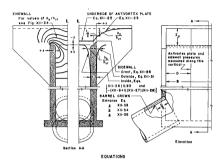
Graave - edged barrel entrance

$$K_1 = 0.25 + \frac{0.55}{(8/0)^{9/6}} - 0.255$$
 (XII-13)

Entrance Loss Coefficient

$$K_0 = K_0 \left(\frac{\pi}{48/D} \right)^2 + K_1$$
 (X11-17)

* The quentity in pointed brockets, $\langle \ \rangle$, is zero for negative values. PIGURE XII-29.—Capacity of the two-way drop inlet.



ENDWALL , Interpolate from Fig.XII-24

BARREL GROWN, D/2 inside the barrel for-

Entronce o

$$\frac{h_a}{h_{y0}} = -1.365 + 0.5 \cdot \log_{10} \frac{B}{O} + 0.65 \cdot 5$$
 (XII-33)

where for 8/D > 2.4 , use B/D = 2.4

Enfrance b

$$\frac{h_a}{h_{\rm FB}} = -1.385 + 0.5 \log_{10} \frac{B}{D} + \frac{S}{S}$$
 (XII-34)

where for B/D > 2.4. use B/D • 2.4

Enfrance a

$$\frac{h_0}{h_{\infty}} = -LOI + 0.38 \log_{10} \frac{9}{0} + \left(0.85 - \frac{0.95}{8/0}\right) S$$
 (XII-35)

FIGURE XII-30.-Pressures in the two-way drop inlet.

IINOFRSIDE OF ANTIVORTEX PLATE

Incide Oren Intel

$$\frac{h_{\pi}}{h_{\pi r}} = \frac{0.55 \pm 0.25 \log_{10} \left(t_{c} / 0 \right)}{4 \left(z_{\pi} / 0 \right)^{\frac{1}{2}}} \tag{XII-25}$$

Outside Oren Inlet

$$\frac{h_{p_1}}{h_{p_2}} = -\frac{1}{\left(2 \, Z_p / D\right)^2} 16 \, q_{\infty}^{-1} \left[-0.125 - \frac{0.44}{Z_p / D} \, \frac{x}{D} \right] \tag{XII-23}$$

SIDEWALL Crest

$$\frac{h_{n}}{h_{Vf}} = \left\{4.5 \frac{I_{0}}{O} - 4.25\right\} + \frac{1}{Z_{p}/O} \left[2 - \frac{1.6}{(Z_{p}/O)^{1/3}}\right] \tag{X11-26}$$

where {4.5 10 - 4.25} s -2

Inside Oron Inlat

For 1/0-0.25,use 0.93 times the pressure on the creat given by Eq. XII-26 For 0.5 5 1/0 5 2 8

where the:

Creet less ocefficient, K.

$$K_0 = \sqrt{\left(-2\frac{t_0^{-1}}{0}\right)^8 + \frac{0.1}{(Z_1/\Omega)^8} + 0.02\left(\frac{8}{0}\right)^{8/6}}$$
 (XII-9)

where $\left\langle 1-2\frac{t_0}{0}\right\rangle = 0$ for negative values

Correction for drop inlet length , K B / D

$$K_{B/D} = \frac{1.742 - \log_{10}(z/O)}{1.3344} b^{-(B/D)^2}$$
(XII-27)

where for B/O>7, use 5/O-7; and for 1/O = 0.5 , 0.75 , 1.0 , 1.5 , 2.0 , 2.5 b = 1.0145, 1.0125, 1.0112, 1.0090, 1.0047, 1.0000

Correction for great thickness, K. . . .

$$K_{1_{g}/D} = \left(0.68 + 0.08 \frac{2}{D}\right) \left(\frac{1_{g}}{D}\right)^{-\left(0.674 - 0.678 \frac{2}{D}\right)}$$
(XII - 28)

where 0.5 \$ 2/0 \$ 2.5 Outside Orap Inlet

$$\frac{h_0}{h_{17}} = -\frac{1}{32(Z_0/G)(z/G)}$$
(XII-31)

Figure XII-30.-Continued.

Nomenclature

- a s constant, dimensionless

 A, area of barrel, in square feet

 A. area of drop inlet or riser, in square feet
- b a constant, dimensionless
- B drop inlet length, in feet C discharge coefficient in equation I-1, in En-
- glish units

 Contraction coefficient for the jet in the drop
 inlet, dimensionless
- D barrel diameter and drop inlet width, in feet

 Di width of the jet in the drop inlet, in feet
- f Darcy-Weisbach friction factor in the barrel,
- f. Darcy-Weisbach friction factor in the drop inlet or riser, dimensionless gravitational acceleration, in feet per second
- per second

 h. head loss at the drop inlet crest, in feet

 h. total head loss for the drop inlet, in feet
- in the barrel, the local pressure head deviation from the friction gradeline, in feet; in the drop inlet, the local pressure head relative to the pressure at the same elevation outside the drop injet; in feet
- b_{oc} the pressure head on the drop inlet crest relative to the pressure in the reservoir at crest elevation, in feet
- head lose caused by the barrel entrance or transition, in feet
 welcoity head between the antivortex plate
- and the drop inlet crest $= V_e^2 / 2 g$, in feet h,p velocity head in the barrel $= V_p^2 / 2g$, in
- welocity head in the drop inlet = V,2 / 2 g, in
- H head on crest, in feet
 H total head from water surface to point at
 - which the hydraulic gradeline pierces the plane of the outlet = $H + Z + \left(\frac{D}{2} - \beta D\right) \cos^{-1}(\sin S)$,
- or to the tailwater surface, in feet

 Karo multiplier correction for drop inlet length.
- dimensionless

 K. creat loss coefficient, dimensionless

 K. Borda mouthpiece creat loss coefficient, fictitious creat loss coefficient for an infinite
- plate height based on the outside width of the drop inlet, dimensionless K. entrance loss coefficient, dimensionless
- K. ontist loss coefficient, dimensionless

- K_t berrel entrance loss coefficient, dimensionless K_{ton} multiplier correction for crest thickness, di-
- mensionless

 t length of conduit, in feet
 - t. length of drop inlet or riser, in feet L crest length, in feet
 - L, antivortex plate overhang, in feet
 - n a specific point in the spillway

 O discharge, in cubic feet per second
- radius of inside corner of crest, in feet
- R. hydraulic radius of drop inlet or riser, in feet
 R. Reynolds' number, dimensionless
- S conduit slope, sine t. crest thickness, in feet
- V. velocity between the antivortex plate and
 - the drop inlet crest, in feet per second

 (i) velocity of the jet in the drop inlet, in feet

 per second
 - V_p velocity in the barrel, in feet per second
 V_c average velocity in the riser or drop inlet. in
 - feet per second

 V., s fictitious average velocity in drop inlet based
 on the outside drop inlet width, in feet per
 - second
 w unit weight of water, in pounds per cubic
 foot
- W drop inlet width, in feet
 x distance from outside wall of drop inlet, in
- feet

 vertical distance below drop inlet crest, in
 feet, x is positive below and negative above
 - the crest elevation.

 difference in elevation between inlet crest or conduit invert at inlet and centerline of
- outlet, in feet

 Z, height of drop inlet, crest to invert of barrel
 at entrance, in feet
- at entrance, in feet
 beight of antivortex plate above the drop inlet crest, in feet
- Δp. difference between the pressure at any point in the drop inlet and the static pressure outside the drop inlet at the same elevation, in pounds per square foot
- β ratio of the distance above the invert at which the projected hydraulic gradeline pierces the plane of the conduit exit to the conduit dismeter, dimensionless
- kinematic viscosity, in square feet per second surface tension, in dynes per centimeter the quantity in the pointed brackets is zero
 - for negative numbers

